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1 Modeling the hygrothermal behavior of green walls in Comsol Multiphysics®: Validation
2 against measurements in a climate chamber

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33 **Abstract**

34 Green walls (GW) can diminish building's surface temperature through shading, insulation, and
35 evapotranspiration mechanisms. These can be analyzed by computer models that account for
36 heat and mass transfer phenomena. However, most previous models were one-dimensional
37 thermal simulations in which boundary conditions (BC), like convective moisture transport,
38 were not or only partly considered. The present work proposes a more comprehensive way to
39 predict GW's hygrothermal behavior by integrating a 3D multiphysics model that couples heat
40 and moisture transport in Comsol Multiphysics[®]. The air cavity that usually separates the GW
41 from the building was also considered. Heat sink terms were added to represent plants'
42 transpiration and substrates' evaporation, considering the leaf area density (LAD) and
43 substrate's water saturation (Sr). The model was validated against experiments where four
44 green wall-test panels (GW-TPs) were evaluated in a climate chamber under steady-state
45 conditions. This provides a much sounder approach for validation than what currently exists (r
46 $= 0.97$; $RMSE = 0.33$ °C). The four GW-TPs decreased the masonry's surface temperature in
47 the range of 0.89 to 1.14 °C (0.97 ± 0.11 SD °C). The average contribution of the
48 evapotranspiration effect was 30%, whereas the contribution of the air cavity was 60.7 ± 0.09
49 %. The temperature at the substrate's rear was reduced on average by 0.57 ± 0.15 SD °C. When
50 solar radiation was considered as a BC, the GW-TPs decreased the building's surface
51 temperature by 10°C. Lastly, high values of LAD and Sr translated into increased temperature
52 reduction values.

53 **Keywords:** evapotranspiration, green wall, heat transfer, hygrothermal behavior, mass
54 transfer, multiphysics modeling.

55

NOMENCLATURE

BC	Boundary condition	(-)
C	Concentration of moisture	kg m ⁻³
C_p	Heat capacity at constant pressure	J kg ⁻¹ K ⁻¹
c₀	Empirical coefficient	(-)
c₁	Empirical coefficient	(-)
c₂	Empirical coefficient	(-)
D	Vapor diffusion coefficient of air	m ² s ⁻¹
d₀	Displacement height	(-)
ET_o	Evapotranspiration of reference	mm day ⁻¹
F	Volume force	N m ⁻³
G	Global moisture source	kg m ⁻³ s ⁻¹
g	Gravity	m s ⁻²
g_w	Moisture vector flux	kg m ⁻² s ⁻¹
GW	Green wall	(-)
GW-TP	Green wall-test PANEL(s)	(-)
h_{ht}	Heat transfer coefficient	W m ⁻² K ⁻¹
h_m	Mass transfer coefficient	m s ⁻¹
k	Thermal conductivity	W m ⁻¹ K ⁻¹
K_c	Active leaf area coefficient	(-)
L	Plate length	m
LAD	Leaf area density	m ² m ⁻³
m	Mass vector flux	kg m ⁻² s ⁻¹
p	Vapor pressure	Pa
Q	Global heat source	W m ⁻³
Q_{trans}	Heat sink transpiration (vegetation)	W m ⁻³
Q_{ev}	Heat sink evaporation (substrate)	W m ⁻³
q	Heat vector flux	W m ⁻²
r_a	Aerodynamic resistance	s m ⁻¹
Ra_L	Rayleigh number	(-)
RH	Relative humidity	(%)
r_s	Stomatal resistance	s m ⁻¹
r_{sub}	Resistance to vapor transfer	s m ⁻¹
Str	Substrate's water saturation	m ⁻³ m ⁻³
T	Temperature	K
u	Wind speed	m s ⁻¹
w	Water content	kg m ⁻³
y	Vegetation thickness	m
z	Wind speed altitude measurements	(-)
z_{om}	Roughness length	(-)
GREEK SYMBOLS		
Δ	Slope of the saturated vapor pressure curve	kPa K ⁻¹
α_v	Volumetric coefficient of thermal expansion	(-)
δ	Vapor permeability of still air	s
δ_p	Vapor permeability	s
φ_w	Relative humidity	(-)
γ	Thermodynamic psychometric constant	Pa K ⁻¹
κ	Von Karman's constant	(-)
η	Dynamic viscosity	Pa s
μ	Vapor resistance factor	(-)
ρ	Density	kg m ⁻³
SUBSCRIPTS		
a	Aerodynamic	
air	Air	
conv	Convective	
eff	Effective	
ext	Exterior	
gb	Rear substrate	
ht	Heat transfer	
int	Interior	
m	Mass transfer	
ref	Reference	
s	Stomatal	
sat	Saturation	
solid	Solid	
sub	Of the substrate	
surf	Surface	
w	Outer masonry's surface	

58 **1. Introduction**

59 1.1 Green walls and urban heat island effect

60 Rapid urbanization has caused urban heat islands (UHI) to become a common problem in cities
61 worldwide. As a result, higher urban temperatures lead to discomfort, impact human health,
62 and increase energy consumption for cooling (Koch *et al.*, 2020). Increasing the amount of
63 vegetation in cities has proven to be an effective mitigation measure to reduce the UHI effect.
64 Green walls (GW) are a kind of urban vegetation technology that may be used for this purpose.
65 They can be classified based on the growing type into "green faces or vegetated coverings"
66 (GF) and "living wall systems" (LWS) (van de Wouw, Ros and Brouwers, 2017). GF and LWS
67 have been researched at different scales (Malys, Musy and Inard, 2014).

68 The associated benefits of GW include energy savings, urban microclimate regulation, sound
69 attenuation, air purification, and social and psychological aspects (Djedjig *et al.*, 2017). Several
70 studies have demonstrated that green walls can decrease the indoor temperature from 0.5 to 7
71 °C, depending on the type of green wall and preconditions such as orientation (Malys, Musy
72 and Inard, 2014). The thermal behavior of GW has been described as very sensitive to the
73 climatic context, characteristics of the green coating, and operational configuration.
74 Consequently, thermal effects are often the focus of green wall studies, which are investigated
75 by comparing the building wall's temperature with and without a vegetated envelope. The
76 thermal behavior of a GW will significantly differ from common construction materials, given
77 that they consist of living materials like plants. In addition, a substrate layer also modifies the
78 thermal behavior, and it serves as attachment and nutrition (Djedjig *et al.*, 2017a).

79 1.2 Hygrothermal benefits of green walls

80 GW's benefits can be investigated from a hygrothermal viewpoint, i.e., heat and moisture
81 transfer, by accounting for the most influential phenomena through its different layers. Three






82 main mechanisms are usually considered: shading, insulation, and evapotranspiration (Djedjig
83 *et al.*, 2012). The first relates to solar radiation, as plants reflect part of the incoming solar
84 radiation depending on their characteristics, transmitting only a fraction to the next layer.
85 Insulation refers to the thermal resistance to heat transfer through the GW's layers, and
86 empirical relations have been developed to calculate heat transfer coefficients between the GW
87 and surroundings (Fabiana Convertino, Vox and Schettini, 2019). Plants have a specific heat
88 capacity similar to that of water but a lower thermal conductivity and can therefore act as
89 efficient heat sinks (Jayalakshmy and Philip, 2010). In addition, an air cavity usually exists
90 between the GW and the building, which can offer additional insulation if poorly ventilated
91 (Convertino, Vox and Schettini, 2021).

92 Research has shown that the most challenging task is analyzing the evapotranspiration effect
93 due to the dynamic behavior of plants. Seasonal and growth cycles and changes in the
94 substrate's moisture contribute to GW's changing aspects. Therefore, modeling
95 evapotranspiration is likewise challenging yet relevant as it is recognized as a cooling tool.
96 Plants consume 2.45 MJ/kg of latent heat to vaporize water which cools the surroundings
97 adiabatically (van de Wouw, Ros and Brouwers, 2017). Hence, plants act as heat sinks, and the
98 more latent heat they consume translates into more cooling. Plant growth cycle and
99 characteristics, substrate properties, and weather conditions determine evapotranspiration (van
100 de Wouw, Ros and Brouwers, 2017).

101 Traditionally, the thermal modeling of GW involves a heat transfer analysis in which the
102 general heat conduction equation is employed (Šuklje, Medved and Arkar, 2016; Widiastuti *et*
103 *al.*, 2022). Since this is a partial differential equation, boundary conditions corresponding to the
104 outdoor and indoor environment are used to achieve a solution. Frequently, a steady-state
105 calculation is adopted, neglecting transient effects (Carlini *et al.*, 2014). Nevertheless, the
106 reality is more complex since outdoor and indoor conditions are transient. The latter implies

107 that high interest in analyzing time-dependent hygrothermal behavior. On the other hand,
 108 hygrothermal models have been scarcely developed in which moisture transport is considered
 109 (Table 1). Previous thermal and hygrothermal models of GW adapt equations from green roof
 110 models using balances applied to the external and internal faces of the GW (Malys, Musy and
 111 Inard, 2014).

112 Table 1. Recent thermal and hygrothermal models of green walls in buildings: their scope and considered mechanisms.

Authors/Year	Software Integration	Model's scope		Green walls' benefits simulated			Indoor conditions
		Thermal	Hygro-thermal	Shading	Insulation	Evapo-transpiration	
(Arengi, Perra and Caffi, 2021)	EnergyPlus 	✓		YES	YES	YES	NO
(Freewan, Jaradat and Amaireh, 2022)	DesignBuilder 	✓		YES	YES	YES	NO
(Zhang, Zhang and Meng, 2022)	EnergyPlus 	✓		YES	YES	YES	NO
(Kenai et al., 2021)	TRNSYS 	✓		YES	YES	NO	YES
(Škerget, Tadeu and Almeida, 2021)	Finite Difference Method (FDM)		✓	YES	YES	YES	YES
(Hoffmann et al., 2021)	R 		✓	YES	YES	YES	YES

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114 1.3 Objectives and scope of the study

115 As seen in **Table 1**, previous work on GW modeling has primarily been thermal-based,
116 employing a one-dimensional nodal approach. These models have been integrated into common
117 Building Energy Simulation software like TRNSYS and EnergyPlus (Djedjig, Bozonnet and
118 Belarbi, 2016; Arengi, Perra and Caffi, 2021). The nodal approach requires fewer calculations,
119 as meshing is not necessary. Nevertheless, it treats each zone as a homogenous volume with
120 uniform physical variables. For instance, the temperature distribution is not obtained within a
121 volume (Hamdaoui *et al.*, 2021).

122 Certainly, thermal models of GW have neglected the convective moisture transfer (which
123 affects heat transfer) occurring at the external surface of the GW's vegetation, i.e., exterior, and
124 the internal surface of the building's, i.e., interior. Therefore, both heat and moisture transfer
125 phenomena must be coupled in a hygrothermal model of GW to approximate reality. For this
126 purpose, the vapor resistance factor must be considered as it determines the moisture transfer.

127 The present study aims to model the hygrothermal behavior of green walls (GW) mounted on
128 the external surface of buildings by integrating and coupling heat and moisture balance
129 equations into a three-dimensional (3D) multiphysics model. The software package Comsol
130 Multiphysics[®] is utilized for GW for the first time. The CFD approach provides a detailed
131 analysis of the various fluxes occurring inside and outside buildings, which is desired for a
132 hygrothermal model of a GW. Yet it takes more time to converge to a solution than a nodal
133 approach; it provides detailed information on the temperature field at all points in space
134 (Hamdaoui *et al.*, 2021). Therefore, the 3D geometry was chosen to better represent a GW's real
135 dimensions and apply more boundary conditions.

136 Additionally, a 3D geometry offers the advantage of modeling the air cavity between the GW
137 and the building wall and calculating its contribution to the reduction in temperature at the

138 building's wall surface. Additional sink heat terms for the transpiration (plants) and evaporation
 139 (substrate) mechanisms are introduced to account for the evapotranspiration effect of a GW.
 140 Two key parameters of a GW were varied to analyze the effect on temperature reduction,
 141 namely leaf area density (**LAD**) and substrate's water saturation (**Sr**). The model output was
 142 validated against experimental data from four commercial green wall-test panels (GW-TPs)
 143 evaluated in a climate chamber.

144 2 Experiments in a climate chamber

145 2.1 Green wall-test panels

146 Four commercial green wall-test panels (GW-TP) or segments of actual green walls (**Appendix**
 147 **A**) consisting of two compartments, (i) vegetation and (ii) substrate, were evaluated in a climate
 148 chamber. Each GW-TP contained a different substrate and the same combination of plant
 149 species. An overview of the GW-TP and the CW construction is shown in **Table 2**.

150 *Table 2. Geometry of the setup reproduced in the climate chamber with a 6 cm-air cavity between the green wall-test panel*
 151 *(GW-TP) and the outer leaf of the insulated cavity wall (CW).*

Section	Layer	Green wall-test panel (GW-TP)	Thickness (m)	Composition
1. Green wall (GW)	Vegetation	All	0.05	<i>Bergenia cardifolia, Carex morrowii, Vinca minor, Lonicera nitida, Iberis sempervirens, Lavandula angustifolia</i> (randomly planted)
	Substrate	A	0.10	Rockwool
		B	0.05	Rockwool
		C	0.10	Potting soil
		D	0.10	Peatmoss
2. Air cavity (AC)	Air	-	0.06	-
3. Insulated air cavity wall (CW)	Masonry	-	0.10	
	Air		0.03	
	Rockwool		0.05	-
	Thermobrick		0.09	

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156 2.2 Climate chamber

157 A modular two-zone climate chamber was used to evaluate the GW-TP (**Appendix B**). The
158 climate chamber allows recreating outdoor and indoor conditions on both sides of a green wall-
159 test panel (GW-TP) by stationary or dynamic input of temperature and relative humidity. These
160 settings were used as boundary conditions in the hygrothermal model (**section 3.3**). The climate
161 chamber applied stationary temperature and relative humidity setpoints to reproduce conditions
162 of a typical summer day in Belgium at noon (exterior = 24°C T, 55% RH; interior: 21°C T, 45%
163 RH). Notably, solar radiation was not reproduced in the climate chamber due to the absence of
164 an artificial sun.

165 An air cavity exists between the backside of the green wall-test panel (GW-TP) and the outer
166 leaf of the insulated cavity wall (CW), i.e., the masonry's surface. For all GW-TPs, the air cavity
167 was enclosed to avoid intense mixing with the air in the room to mimic the real conditions
168 behind a large continuous green wall. A cavity distance of 6 cm was maintained for all the
169 evaluated GW-TPs. Furthermore, sensors to monitor temperature (T1-T6) (FPA22L0100
170 ALMEMO® ± 0.2 K) and relative humidity (RH1-RH6) (FHAD4641L05 ALMEMO® ± 3%).
171 They were purposefully installed across the setup in three regions of interest: (i) in the middle
172 of the vegetation, (ii) on the backside of the substrate, and (iii) at various depths in the
173 construction of the insulated cavity wall (CW). Sensors were also installed in the exterior and
174 interior rooms of the chamber (**Appendix B**). Therefore, temperature and relative humidity
175 profiles across the setup could be obtained to be contrasted against the hygrothermal model.

176 **3 Hygrothermal model**

177 3.1 Geometry

178 The geometry was first defined to build the model in Comsol Multiphysics®. The model's
179 geometry replicated the green wall section (GW-TP) and the insulated cavity wall (CW)
180 installed in the climate chamber (**Table 2**). Then, a mesh convergence study was performed

181 considering five meshing options. The comparison was based on the model's output, i.e.,
182 temperature profiles throughout the geometry, from which datasets containing the points were
183 obtained. A one-way ANOVA was carried out, resulting in a P-value > 0.05 . Therefore, it was
184 concluded that there was no difference among the meshes, and therefore, a normal mesh was
185 used, given its low computational demand.

186 3.2 Model description

187 The hygrothermal model consisted of a steady-state study coupling heat and moisture transfer
188 equations and heat sink terms. The governing equations and boundary conditions are shown in
189 **Table 3**. Data from the external and internal sides of the geometry were included as boundary
190 conditions (**section 2.2**), whereas the rest of the boundaries were adiabatic, i.e., thermally
191 insulated. The vegetation and substrate layers of the GW-TP and the layers constituting the
192 insulated cavity wall (CW) were modeled as homogeneous solid media. An exception was made
193 for the air cavity, which was modeled as a fluid. Layers' thermophysical properties are shown
194 in **Appendix C**. Transpiration and evaporation terms were modeled as uniform volumetric heat
195 sinks. The model was integrated into the commercial software package Comsol Multiphysics®
196 6.0 using a computer with a 64-bit operating system and Intel ® Xeon ® CPU E5-1650 v3
197 @3.50 GHz and 128 GB of RAM.

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Table 3. Summary of the hygrothermal model, including governing equations and boundary conditions. Elaborated from Djedjig et al. (2012, 2017); Cascione et al. (2017); van de Wouw, Ros and Brouwers (2017); Convertino, Vox and Schettini (2019).

PROCESS	No.	NAME	EQUATION	DESCRIPTION	UNITS
A. Governing Equations					
Heat transfer balance	Eq. 1	Heat source	$\nabla \cdot \mathbf{q} = Q - (Q_{\text{trans}} + Q_{\text{evap}})$	The heat of transpiration and evaporation is deducted from the global heat source.	W m^{-3}
	Eq. 2	Heat flow vector field	$\mathbf{q} = -k_{\text{eff}} \nabla T$	Heat transfer depends on effective thermal conductivity.	W m^{-2}
	Eq. 3	Effective thermal conductivity	$k_{\text{eff}} = k_{\text{solid}} \left(1 + \frac{w(\phi_w)}{\rho_{\text{solid}}} \right)$	The effective thermal conductivity is affected by moisture transport.	$\text{W m}^{-1} \text{K}^{-1}$
Moisture transfer balance	Eq. 4	Moisture source	$\nabla \cdot \mathbf{g}_w = G$	The global moisture source is the divergence of the moisture vector field.	$\text{kg m}^{-3} \text{s}^{-1}$
	Eq. 5	Moisture flow vector field	$\mathbf{g}_w = -(\delta_p \nabla(\phi_w p_{\text{sat}}))$	Moisture transfer is given by the material's permeability and partial pressure of air.	$\text{kg m}^{-2} \text{s}^{-1}$
	Eq. 6	Material's vapor permeability	$\delta_p = \frac{\delta}{\mu}$	The vapor resistance factor gives the material air permeability.	s
B. Boundary Conditions					
Convective heat transfer	Eq. 7	Convective heat flux	$q_{\text{conv}} = h_{\text{ht,conv}}(T_{\text{surf}} - T_{\text{ext}})$	Convective heat transfer depends on a heat transfer coefficient and a temperature gradient.	W m^{-2}
	Eq. 8	Convective heat transfer coefficient (exterior)	$h_{\text{ht,conv,ext}} = 5.9 + 4.1u \frac{511 + 294}{511 + T_{\text{ext}}}$	Empirical formulas describe the convective heat transfer coefficients at the exterior of a green wall and interior.	$\text{W m}^{-2} \text{K}^{-1}$
	Eq. 9	Convective heat transfer coefficient (interior)	$h_{\text{ht,conv,int}} = \frac{k}{L} \left(0.68 + \frac{0.67 \text{Ra}_L^{\frac{1}{4}}}{\left(1 + \left(\frac{0.492k}{\eta C_p} \right)^{\frac{9}{16}} \right)^{\frac{1}{9}}} \right)$		$\text{W m}^{-2} \text{K}^{-1}$
Convective moisture transfer	Eq. 10	Convective moisture flux	$m_{\text{conv}} = h_{\text{m,conv}}(C_{\text{surf}} - C_{\text{ext}})$	Convective moisture transfer depends on a moisture transfer coefficient and a moisture gradient.	$\text{kg m}^{-2} \text{s}^{-1}$
	Eq. 11	Convective moisture transfer coefficient (exterior)	$h_{\text{m,conv,ext}} = \frac{D}{k} \left(\frac{k}{\rho C_p D} \right)^{1/3} * h_{\text{ht,conv,ext}}$	Empirical formulas describe the convective moisture transfer coefficients at the exterior of a green wall and interior.	m s^{-1}
	Eq. 12	Convective moisture transfer coefficient (interior)	$h_{\text{m,conv,int}} = \frac{D}{k} \left(\frac{k}{\rho C_p D} \right)^{1/3} * h_{\text{ht,conv,int}}$		m s^{-1}
Evapotranspiration (heat sink)	Eq. 13	Heat of transpiration-plants	$Q_{\text{trans}} = K_c L A D \frac{(\rho C_p)_{\text{air}}}{\gamma(r_a + r_s)} (p_{\text{sat}} - p_{\text{air}})$	Transpiration heat sink depends on leaf area, resistances, and a vapor pressure gradient.	W m^{-3}
	Eq. 14	Heat of evaporation - substrate	$Q_{\text{evap}} = \frac{(\rho C_p)_{\text{air}}}{\gamma(r_{\text{sub}})} (p_{\text{sat}} - p_{\text{air}})$	Evaporation (heat sink) depends on resistances and a vapor pressure gradient.	W m^{-3}

209), **Eq. 1** and **Eq. 4** show the global heat and moisture balances steady-state conditions. **Eq. 1**
210 describes that the divergence of the heat flow vector field (q) equals the global heat source (Q)
211 from which an amount of heat is deducted by the heat sink terms that correspond to vegetation's
212 transpiration (Q_{trans}) and substrate's evaporation (Q_{ev}), calculated from **Eq. 13** **Eq. 14**
213 respectively. q is given by the effective thermal conductivity (k_{eff}) and the temperature
214 gradient (∇T) (**Eq. 2**). The heat sink terms are introduced as user-defined volumetric heat sink
215 terms and are further elaborated in **section 3.5**

216 The water content depends on relative humidity ($w(\phi_w)$) and affects q by producing a change
217 in effective thermal conductivity (k_{eff}). The latter considers the material's dry solid thermal
218 conductivity (k_{solid}) and density (ρ_{solid}) (**Eq. 3**).

219 The global moisture source (G) in **Eq. 4** is given by the divergence of the moisture transport
220 field vector (g_w) (**Eq. 5**). The model considers vapor diffusion through materials depending on
221 the vapor permeability (δ_p). The materials' vapor resistance factor (μ) was used instead of the
222 vapor permeability (δ_p) and its conversion is automatically performed by considering the vapor
223 permeability of still air (δ) (**Eq. 6**).

224 Natural convection occurring in the enclosed air cavity, i.e., between the green wall test panel
225 (GW-TP) and the outer leaf of the insulated cavity wall, i.e., masonry's surface, was modeled.
226 The air cavity was modeled using the Boussinesq approximation, which states that buoyancy
227 occurs due to a variation in density expressed by temperature differences. A small change in
228 density is accounted for in a volume force term. The latter term is introduced in the momentum
229 equation in the opposite direction of gravity.

230 3.3 Boundary conditions

231 Convection is a common boundary condition when modeling heat and mass transfer, i.e., a fluid
232 cools or heats a surface through natural or forced convection. Forced heat and mass convection

233 are typically the prevailing convection type at the exterior of a green wall, i.e., vegetation's
234 surface (Fabiana Convertino, Vox and Schettini, 2019). In contrast, natural heat and mass
235 convection were assumed at the interior of the insulated cavity wall, i.e., the thermobrick
236 surface. Adiabatic BC was used in the rest of the boundaries, i.e., no heat flux.

237 The forced heat and mass convection BC was introduced by applying a convective heat flux
238 (q_{conv}) and a convective moisture flux (m_{conv}) at the vegetation surface (**Eq. 7 and Eq. 10**).

239 These fluxes depend on the gradient between the bulk temperature (T_{ext}) and bulk moisture
240 (C_{ext}) surrounding the surface and the same quantities at the surface (T_{surf}), (C_{surf}).

241 Convective heat and moisture transfer coefficients are required for the calculation of the fluxes.

242 We employed the empirical relationship developed by Convertino, Vox and Schettini (2019)

243 and Ayata, Tabares-Velasco and Srebric (2011) for the calculation of the convective heat

244 transfer coefficient ($h_{ht,conv,ext}$) at the vegetation surface (**Eq. 8**). On the other hand, empirical

245 relationships available in the software package were used to obtain the convective heat transfer

246 coefficient at the interior ($h_{ht,conv,int}$) (**Eq. 9**), as well as for the moisture transfer coefficients

247 at the vegetation surface ($h_{m,conv,ext}$) (**Eq. 11**) and interior ($h_{m,conv,int}$) (**Eq. 12**). Moisture

248 transfer coefficients calculation depend on the heat transfer coefficient. These relationships

249 involve the Rayleigh number (Ra_L), plate length (L), bulk temperature (T_{ext}), thermal

250 conductivity (k), dynamic viscosity (η), and the vapor diffusion coefficient of air (D).

251 3.4 Thermophysical properties

252 The relevant thermophysical properties in this study include density (ρ), specific heat capacity

253 (C_p), thermal conductivity (k), and a vapor resistance factor (μ), as shown in **Appendix C**. A

254 literature review provided average values for the vegetation and substrate properties. Similarly,

255 the properties of the materials constituting the insulated cavity wall (CW) were obtained from

256 the manufacturer's data sheets.

257 3.5 Evapotranspiration

258 As stated above, the model elaborates and applies heat sink terms, i.e., the heat taken by
259 transpiration (plants) and evaporation (substrate) that is subtracted from the global heat source
260 (**Eq. 13 and Eq. 14**). The two heat sink terms represent the evapotranspiration effect
261 corresponding to the conversion from sensible to latent heat.

262 3.6.1 Plants' transpiration

263 The transpiration heat sink term is an adaptation of the Penman-Monteith equation. This
264 equation provides an ideal reference evapotranspiration value for "a hypothetical reference crop
265 with an assumed crop height of 0.12 m, a fixed surface resistance of 70 s m^{-1} , and an albedo of
266 0.23" (Widiastuti *et al.*, 2022). A modification of the Penman-Monteith equation is necessary to
267 apply it in the hygrothermal model. For this purpose, the leaf area density of the vegetation
268 compartment (LAD) is considered in the calculation of heat of transpiration (Q_{trans}) (**Eq. 13**).
269 LAD values of 0.5, 0.8, 0.6, and 1.0 for the PANEL's A, B, C, and D were visually approached
270 based on the leaf area that covered the substrate. Nevertheless, not all the LAD is active, and
271 hence, an active leaf area coefficient (K_c) was introduced, taking a mean value of 0.9 from
272 previously used values to calculate the effect under adjusted conditions (ET-adj). K_c values
273 ranging 0.1 to 1.7 were reported by Lazzara and Rana (2010) and van de Wouw, Ros and
274 Brouwers, (2017).

275 **Eq. 13** shows the calculation of the transpiration heat sink term, in which $(\rho c_p)_{air}$ represents
276 the air volumetric heat capacity, γ is the thermodynamic psychrometric constant, and r_a and r_s
277 account for an aerodynamic and stomatal resistance to transpiration. p_{sat} is the saturation vapor
278 pressure evaluated at leaf temperature, i.e., vegetation surface, while p_{air} is the vapor pressure
279 of the surrounding bulk air (Arkebauer, 2005).

280 An aerodynamic resistance (r_a) to transpiration was estimated (23 s m^{-1}), expressed in terms of
281 the wind speed (u) and empirical terms. On the other hand, the stomatal resistance to
282 transpiration (r_s) is a purely biological parameter, and a mean value of 150 s m^{-1} was chosen,
283 as reported by Sailor (2008).

284 3.6.2 Substrate's evaporation

285 The heat taken by the substrate's evaporation (Q_{ev}) (**Eq. 14**) is calculated by a difference in
286 saturation vapor pressure, i.e., between the substrate's surface and the surrounding air. A
287 resistance to vapor transfer (r_{sub}) was accounted for, which is given by the substrate's water
288 saturation. An average value of $0.2 \text{ m}^3 \text{ m}^{-3}$ of a typical soil was chosen for all GW-TPs except
289 PANEL D ($0.4 \text{ m}^3 \text{ m}^{-3}$), as the substrate (peat moss) can hold more water (Arengi, Perra and
290 Caffi, 2021).

291 4 Model validation

292 The model's accuracy was assessed using the Pearson correlation coefficient (r) and the root-
293 mean-square error (RMSE). The correlation coefficient (r) describes the association between
294 the predicted and the experimentally obtained values, whereas RMSE assesses the prediction
295 accuracy. On the other hand, the overall evaluation of the benefits of the green walls was done
296 by comparing the results between a bare wall case (section 0), i.e., only insulated air cavity
297 wall, vs. a green wall case, i.e., with GW-TP (section 5.2).

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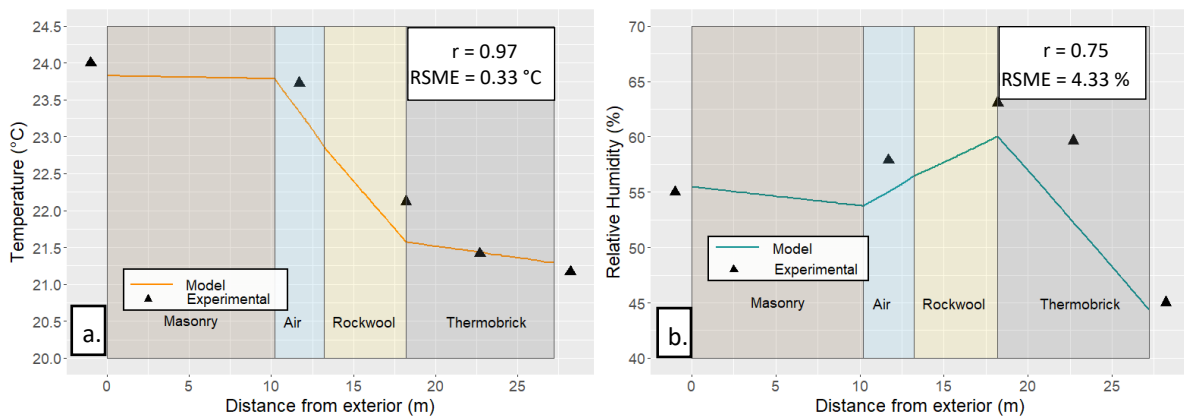
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303 **5 Results and discussion**

304 **5.1 Bare wall scenario**

305 The validation of the hygrothermal model under stationary conditions for the bare wall case,
306 i.e., insulated cavity wall (CW), is shown in **Figure 1**. The bare wall scenario depicts the heat
307 and moisture transfer through the different materials of the insulated cavity wall (CW).
308 Temperature and relative humidity profiles across the insulated cavity wall (CW) were obtained
309 from the model's output and plotted against experimental data from the sensors to validate their
310 congruency. The masonry's surface temperature (T_w) evaluated in this case is the comparison
311 basis for calculating the temperature reduction that the GW provide. A high agreement ($r =$
312 0.97) between the measured points and the model's output was observed for the variable
313 temperature (**Figure 1**). In contrast, the same measure was lower for relative humidity. RMSE
314 was significantly low for temperature (RSME = $0.33\text{ }^\circ\text{C}$) and relative humidity (RSME = $4.33\text{ }\%$)
315 $\%$), indicating a low error.



316

317 *Figure 1. (a) Predicted temperature profile (model) and experimental measurements (triangles) from the insulated cavity wall*
318 *(CW) that represents the bare wall scenario. (b) Predicted relative humidity profile (model) and experimental measurements*
319 *(triangles) for the insulated air cavity wall (CW). Note: The first and the last measured points (outside geometry) in each*
320 *figure represent the exterior and interior room temperature and relative humidity, i.e., setpoints in the climate chamber.*

321 **Figure 1** shows that the masonry provides poor insulation given its thermal conductivity (k
322 $=1.61\text{ W m}^{-1}\text{ K}^{-1}$), which contrasts with the insulation of the internal air gap, i.e., between the
323 masonry and the Rockwool. This air gap was assumed to be a stagnant layer of air ($k = 0.025$

324 $\text{W m}^{-1} \text{K}^{-1}$), and natural convection was not modeled. Consequently, a steep decrease in the
325 temperature profile can be seen in this section, which arises due to the high insulation provided
326 by the next layer of Rockwool ($k=0.035 \text{ W m}^{-1} \text{K}^{-1}$).

327 The hygrothermal model applied to the bare wall scenario predicted a temperature of $23.8 \text{ }^\circ\text{C}$
328 at the masonry's surface.. Notably, the temperature at this point was slightly below the bulk air
329 temperature of the exterior ($24 \text{ }^\circ\text{C}$). The latter can be explained as external forced convection
330 occurring, and no radiation was supplied in the climate chamber. Nevertheless, in real
331 conditions, buildings' outer walls are exposed to solar radiation, and their temperature can
332 exceed ambient air by several units. For instance, Mazzali et al. (2013) evaluated three green walls
333 separated by an open-air cavity from the cladding in a Mediterranean climate. It was found that
334 during sunny days, the temperature difference monitored on the external wall's surface between
335 the bare wall and the wall covered by the GW ranged from $12 \text{ }^\circ\text{C}$ to $20 \text{ }^\circ\text{C}$. On the other hand,
336 the difference reduced to values of $1 - 2 \text{ }^\circ\text{C}$ during cloudy days. A heat flux analysis was
337 performed in which the surface heat transfer coefficient was the combination of the convective
338 and radiative heat transfer coefficients, similar to the approach taken by Pastori et al. (2021).
339 During high solar radiation values, a negative or outgoing flux was identified from the wall.

340 Freewan, Jaradat and Amaireh (2022) integrated and validated a thermal model for green walls
341 on DesignBuilding. A bare wall scenario was considered to evaluate the temperature reduction
342 provided by the green walls. It consisted of concrete (0.3 m) and bricks (0.1 m). The temperature
343 reduction was measured at the exterior surface of the masonry. The model showed high
344 agreement, and the green walls could decrease the masonry's external surface by $7 \text{ }^\circ\text{C}$.
345 Analogously, the thermal model integrated by Zhang, Zhang and Meng (2022) on EnergyPlus
346 also compared a bare wall scenario against a green wall case based on the external surface
347 temperature of the masonry. The bare wall was built according to the Chinese national standards

348 “Code for thermal design of civil engineering”. The model could predict with a high agreement
349 the external surface temperature of the masonry (RMSE = 0.53 °C).

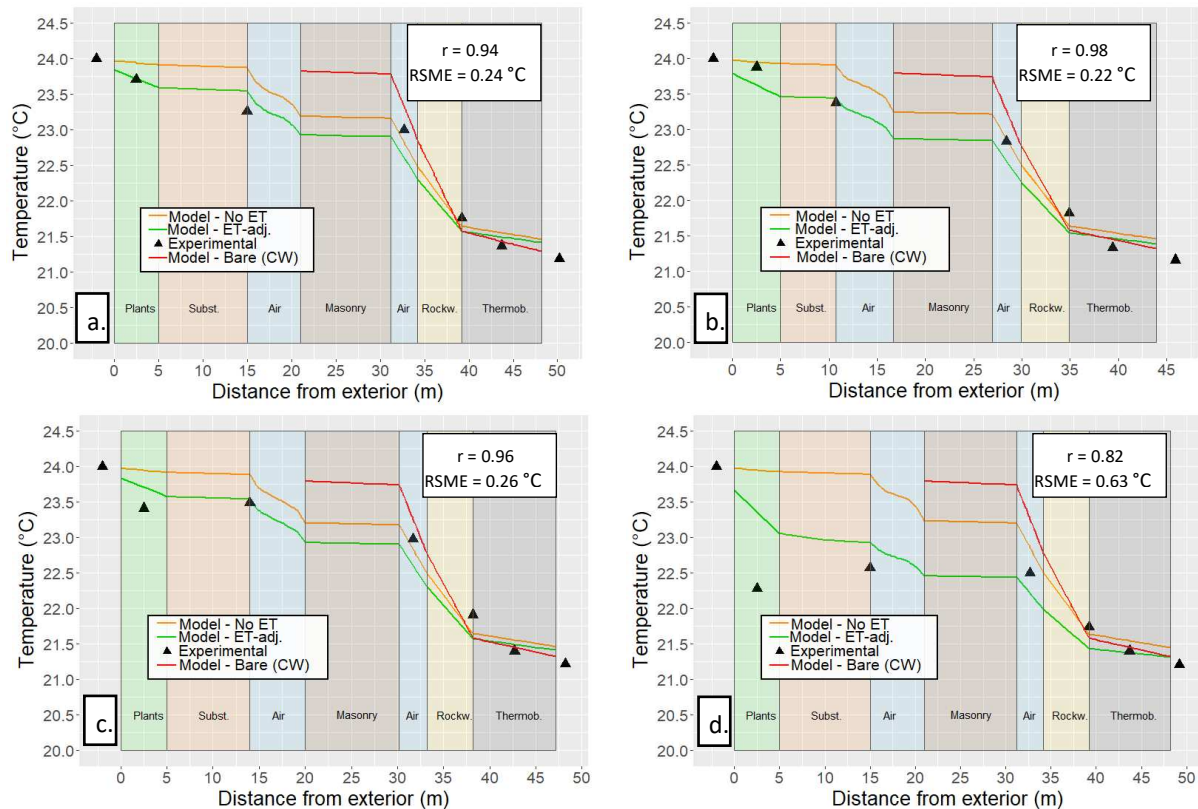
350 The one-dimensional thermal model for green walls built by Scarpa, Mazzali and Peron (2014)
351 considered a bare wall made of concrete (thickness = 0.40 m). It was validated with
352 experimental measurements by considering the external surface wall’s temperature. The model
353 showed high correlation agreement and low values for RMSE, which agrees with our results.
354 García et al. (2019) adapted and integrated two well-known green roof models with EnergyPlus
355 to model heat and mass transfer through green walls and validated them under semiarid climate
356 conditions. A high agreement between experimental and simulated data for the temperature of
357 the foliage and substrate of the green wall was found ($r > 0.80$) and low RSME values (< 2.96
358 °C).

359 5.2 Green wall scenario

360 The temperature reduction offered by the GW-TPs can be seen as the summation of the
361 reduction offered by insulation, evapotranspiration and air cavity mechanisms. Our work
362 analyzes the temperature reduction offered by the GW-TP at the masonry’s surface (T_w) against
363 the bare wall case (**section 0**). This variable has been previously used to measure the cooling
364 ability of green walls, as done by Arengi, Perra and Caffi (2021) and Convertino, Vox and
365 Schettini (2021). However, the hygrothermal model of GW implemented by Malys, Musy and
366 Inard (2014) in SOLENE-microclimate focused on predicting the leaf and substrate temperature
367 of the GW instead of the temperature reduction at the building wall’s surface.

368 **Figure 2** shows the predicted hygrothermal behavior, i.e., temperature profiles, and measured
369 temperature, i.e., points for the four GW-TPs. The validation of the hygrothermal model for
370 each GW-TP is likewise shown as in the bare wall case. A distinction was made between the
371 scenario of no evapotranspiration (No-ET), i.e., no heat sinks, versus evapotranspiration (ET-

372 adj). The latter included the vegetation's transpiration and the substrate's evaporation heat sink
 373 terms. The maximum air velocity obtained in the closed cavity for the four GW-TPs was 0.03
 374 m s^{-1} , which is typical for natural convection. The bare wall case's temperature profile is also
 375 shown as the comparison basis.



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Figure 2. Simulated temperature (profiles) corresponding to different scenarios: (i) No evapotranspiration (No ET); (ii) Evapotranspiration (ET-adj) and experimental data (triangles) for the green wall case, i.e., green wall-test panel (GW-TP) mounted on the outer leaf of the insulated cavity wall (CW). No evapotranspiration (No ET) refers to the no use of heat sinks for evapotranspiration; evapotranspiration (ET-adj) is the scenario where heat sinks are enabled together with the insulation effect and the air cavity. The evaluated panels are (a) PANEL A; (b) PANEL B; (c) PANEL C; (d) PANEL D. Note: The first and the last triangle correspond to the setpoints in the climate chamber.

384 As seen in **Figure 2**, all GW-TPs produced a decreasing temperature profile from the exterior
 385 of the GW, i.e., vegetation's surface to the masonry's surface under both no-evapotranspiration
 386 (No-ET) and evapotranspiration (ET-adj) scenarios. The no-evapotranspiration scenario (No-
 387 ET) is a reference to analyze the extent of the temperature reduction, assuming that the GW-
 388 TPs provide only insulation; thus, the heat sink terms are not enabled. This scenario is important
 389 to comprehend the magnitude of the temperature decrease due to evapotranspiration (**Figure**
 390 **3**). Notably, the temperature reduction was greater under evapotranspiration conditions (ET-
 391 adj), particularly for PANEL B and PANEL D, as their LAD values were the highest among

392 the panels (PANEL B: $LAD = 0.8$; PANEL D: $LAD = 1.0$). Particularly, PANEL D had a thick
 393 substrate with a high water saturation ratio ($Sr = 0.4$) reaching the maximum temperature
 394 reduction amongst the GW-TPs (1.14 °C). Similarly, to our findings, Arengi, Perra and Caffi,
 395 (2021) integrated a hygrothermal model in EnergyPlus and compared the surface temperature
 396 reduction when the GW was present against a concrete model. The study found that the
 397 reduction was up to 13°C due to the high thickness of the substrates that can subsequently host
 398 greater foliar density (LAD). Such results seem to outperform what we found in our work.
 399 Nevertheless, they considered the air cavity ventilated and solar radiation used as a boundary
 400 condition.

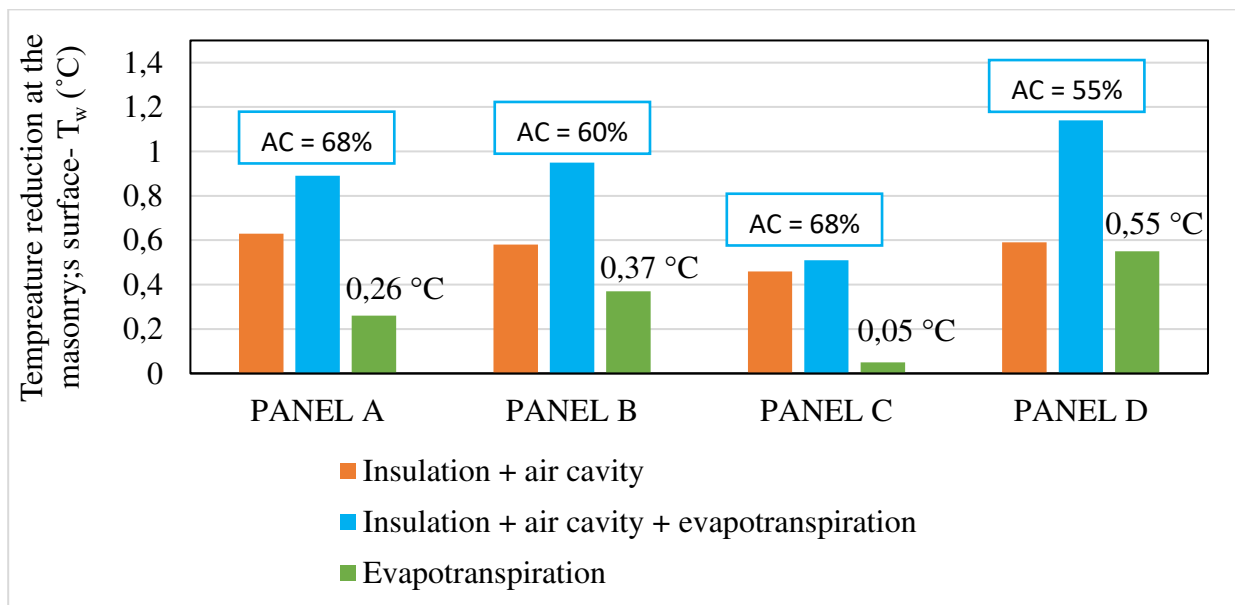
401 The major contribution of the air cavity to the temperature reduction of the masonry's surface
 402 (all GW-TPs) can be seen in **Figure 2**. This indicates that the thermal conductivity of air was
 403 not enhanced to a great extent by modeling natural convection in this section. More details about
 404 the air cavity contribution are given in **Figure 3**. **On the other hand**, the predicted temperature
 405 reduction at the substrate's rear (T_{bg}) of the GW-TPs was compared against experimental
 406 measurements (Fout! Verwijzingsbron niet gevonden.). The mean temperature reduction at the
 407 substrate's rear of the GW-TPs was 0.83 ± 0.41 SD °C compared to the model's output under
 408 evapotranspiration conditions (ET-adj) (0.57 ± 0.15 SD °C).

409 *Table 4. Temperature reduction at the green wall-test panels' substrate rear (T_{gb}) due to the insulation and*
 410 *evapotranspiration effects. Note: This reduction is calculated based on the exterior temperature (24 °C).*

Green Wall – Test Panel (GW-TP)	Temperature reduction at the substrate's rear - T_{bg} (°C)		
	Experimental	Model	Correlation
A	0.74	0.46	r = 0.92
B	0.62	0.56	
C	0.51	0.46	
D	1.43	0.78	

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412 In this study, no experimental measurements were taken at the masonry's surface; thus, the
 413 GW-TPs contribution to reducing the masonry's surface temperature (T_w) under the different
 414 mechanisms is based on the model's output (**Figure 3**). The contribution of the air cavity to
 415 reducing the masonry's surface temperature (T_w) under evapotranspiration conditions was also
 416 calculated. On average, the GW-TPs decreased the masonry's surface temperature (model's
 417 output) by 0.97 ± 0.11 SD °C under the insulation + air cavity + evapotranspiration effects, i.e.,
 418 blue bars in **Figure 3**. The temperature reduction attributed to the evapotranspiration effect
 419 under this scenario was 0.31 ± 0.20 SD °C (mean for all GW=TPs), representing, on average,
 420 30% of the total temperature reduction at the masonry's surface. However, it reached 48% in
 421 the case of PANEL D. On the other hand, the air cavity accounted for 60.7 ± 0.09 % (on average)
 422 of the total temperature reduction at the masonry's surface



423
 424 *Figure 3. Contribution of the different mechanisms to the temperature reduction at the masonry's surface (T_w). AC: Air cavity*
 425 *contribution.*

426 Generally, the building surface's temperature reduction found in this work is significantly lower
 427 than in previous work on green wall modeling. For instance, the thermal model integrated by
 428 Hoffmann et al. (2021) in the software package R simulated 36 scenarios of green cases vs. non-
 429 green cases (bare wall), and the exterior wall temperature reduction in all cases was up to 17

430 °C. Nevertheless, a high LAD value of 6.1 ± 0.5 was considered in all simulations, which is
431 significantly greater than ours.

432 The work of F. Convertino, Vox and Schettini (2019) compared different mathematical methods
433 to calculate the convective heat flux. The calculated convective flux was validated against
434 experimental measurements from a green façade made of evergreen *Pandorea jasminoides*
435 “Variegata” and a control wall (bare wall). In the experimental setup, the layers considered were
436 external air, green layer, air gap (0.15 m), and external and internal surfaces of the building's
437 external wall. The results showed that the empirical expressions efficiently predicted the
438 convective flux in a green wall. However, the authors stated that convective exchanges were
439 not dominant but solar radiation which directly influences evapotranspiration. Their
440 experimental results showed that the reduction reached at the external surface temperature
441 covered by the green façade (compared to the bare wall) from 10 AM to 5 PM was
442 approximately 8 °C.

443 A mathematical model based on a heat balance and considering the shading, insulation, and
444 evapotranspiration effect was integrated with EnergyPlus by Dahanayake and Chow (2017).
445 Notably, the model considered solar radiation as a boundary condition from which the latent
446 heat flux of evapotranspiration was calculated. The simulation showed that the GW could
447 reduce building facades' surface temperature, especially during summer. As a result, a
448 maximum reduction of 26 °C was reached in the exterior surface temperature of the façade.

449 Jim and He (2011) considered incident solar radiation as a boundary condition in a one-
450 dimensional thermal model validated against experimental results. A shading coefficient was
451 used to evaluate the shading performance of the green wall. The study showed that the shading
452 effect of the green wall is dramatic (589.89 W m^{-2} , 39.65 °C) compared with the control wall
453 (1168 W m^{-2} , 48.48 °C). Comparative, Kenai et al. (2021) focused on the radiative and convective

454 exchange between green walls and the external environment. The external or outside surface
455 temperature was likewise used to measure the reduction against a reference or bare house. It
456 was found that under solar radiation peaks ($813\text{-}850\text{ W m}^{-2}$), the external surface temperature
457 did not exceed $45\text{ }^{\circ}\text{C}$ compared to $67\text{ }^{\circ}\text{C}$ in the reference or bare house. Thus, when a thermal
458 model considers solar radiation besides convection, the external surface of a bare wall (T_w)
459 increases significantly. In contrast, a green wall will decrease that temperature as solar radiation
460 promotes more evapotranspiration. Hence, we can hypothesize that the contribution of the GW-
461 TPs in real scenarios might be greater than what we have found in this work.

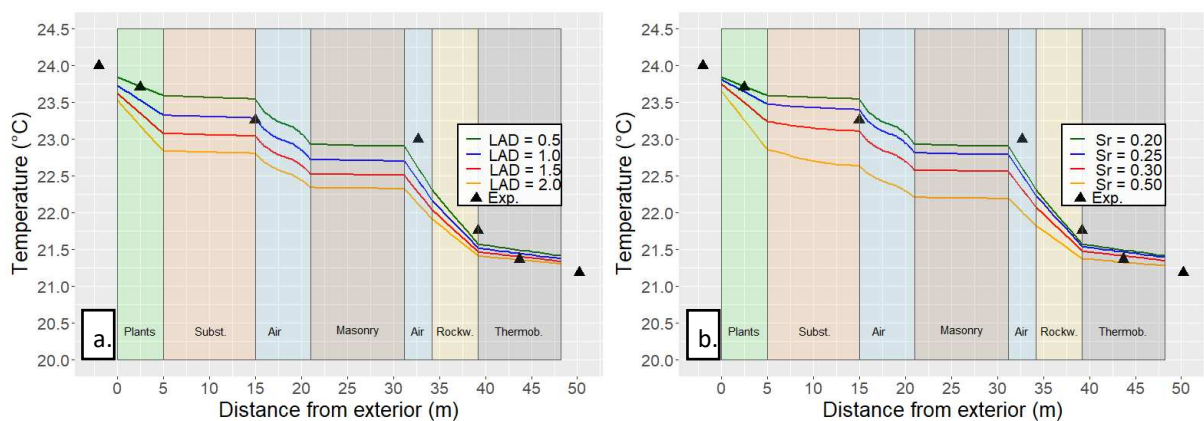
462 A sole simulation was conducted in our software for PANEL 1 to verify the previous hypothesis.
463 Solar radiation (500 W m^{-2}) was imposed as a boundary condition, i.e., heat flux, and the rest
464 of the conditions were maintained constant. The vegetation albedo was set to 0.2 for the green
465 wall scenario, and a value of 0.5 was for the masonry in the bare wall scenario. Our results
466 showed that the masonry's surface temperature (T_w) of the bare wall scenario reached $47.2\text{ }^{\circ}\text{C}$.
467 On the contrary, its value was reduced to $37.6\text{ }^{\circ}\text{C}$ when PANEL 1 was present, i.e., green wall
468 scenario. Hence, a temperature reduction of almost $10\text{ }^{\circ}\text{C}$ was obtained when solar radiation
469 was considered, which agrees with Mazzali et al., 2013 and Arengi, Perra and Caffi (2021).

470 **Figure 3** shows that the air cavity accounts for a major proportion of the temperature reduction
471 at the masonry's surface (T_w), averaging 60% of this reduction when evapotranspiration
472 conditions (ET-adj) were considered. The low thermal conductivity of air can and the low air
473 velocity obtained by modeling natural convection (0.03 m s^{-1}) might explain the high-
474 temperature reduction. Scarpa, Mazzali and Peron (2014) developed and validated a one-
475 dimensional thermal model to compare a green wall mounted on a building wall with an open
476 cavity vs. the same green wall with a closed air cavity. In the latter, natural convection was also
477 modeled similarly to our work. The air cavity in both cases was maintained at 0.05 m thickness.

478 The study showed fewer temperature variations (external wall's surface) were obtained for the
479 green wall with the enclosed cavity, especially during the winter.

480 5.3 Parametric study

481 Crucial parameters in a GW were varied, considering PANEL A to illustrate this impact on the
482 masonry's surface temperature (T_w) (**Figure 4**). Leaf area density (LAD) has been recognized
483 as the most sensitive parameter when calculating vegetation's transpiration. Nevertheless,
484 appropriate techniques are lacking to measure this parameter (De Bock *et al.*, 2022). Therefore,
485 leaf area density (LAD) was varied from 0.50 to 2.00 $m^2 m^{-3}$ and the water saturation ratio (Sr)
486 from 0.20 to 0.40 $m^3 m^{-3}$. Parameter variation aims to identify the most effective combination
487 to obtain the maximum temperature reduction at the masonry's surface.



488

489 *Figure 4. (a) Variation of leaf area density (LAD) and (b) substrate's water saturation (Sr) in PANEL A.*

490 The parametric study showed that high LAD values translated into high-temperature reduction
491 at the masonry's surface (T_w), as the evapotranspiration effect is enhanced by deducting more
492 heat (**Figure 4,a**). For instance, a LAD value of 0.5 $m^2 m^{-3}$ reduced the masonry's temperature
493 (T_w) by 0.9 °C (compared to the bare wall), whereas a value of 2.0 $m^2 m^{-3}$ only by 1.5 °C.

494 Arengi, Perra and Caffi (2021) also performed a parametric study in their green wall's
495 hygrothermal model varying LAD from 0.30 to 5.00 $m^2 m^{-3}$ considering a GW operating under
496 summer conditions. The latter LAD value produced an external surface temperature reduction

497 of 2 °C compared to a LAD of 1 m² m⁻³. Their parametric study also found that high *LAD* values
498 produced a discrepancy between the simulated temperature profiles and the experimental
499 measurements, which is congruent with our results in **Figure 4,a**.

500 Škerget, Tadeu and Almeida (2021) varied the LAD (1.5 – 4.5 m³ m⁻³) and the extinction
501 coefficient to analyze the impact in their hygrothermal model for a green façade (thickness =
502 0.25 m) mounted on a single brick wall without insulation (thickness = 0.20 m). The extinction
503 coefficient refers to the amount of solar radiation decreased by vegetation. The authors found
504 that a dense canopy with a LAD value greater than 3 m³ m⁻³ and an extinction coefficient greater
505 than 0.7 effectively reduced the façade surface temperature.

506 We can hypothesize that the visually approached *LAD* value of PANEL A (0.5 m² m⁻³) might
507 have been the closest to the real value given that if used in the model, the simulated temperature
508 profile matches the experimental data. Nevertheless, this value is conservative since *LAD* values
509 from 2.0 to 3.0 m² m⁻³ have been used for hygrothermal models of GW. Therefore, additional
510 techniques are required to verify the *LAD* values in the GW-TPs.

511 **Figure 4,b** describes that an increase in *Sr* led to a decrease in temperature at the masonry's
512 temperature (*T_w*) since more heat is deducted from the heat balance, i.e., heat sink term. A *Sr*
513 value of 0.2 m³ m⁻³ decreased it by 0.9 °C, whereas a value of 0.25 only by 1.0 °C. We can also
514 maintain the previous assumption of a *Sr* value of 0.2 m³ m⁻³ in all GW-TPs (except for PANEL
515 D, *Sr* = 0.4 m³ m⁻³), as this profile is the closest to the experimental data. It is sustained that
516 variations in the water content of the substrate significantly contribute to temperature reduction.

517 Malys, Musy and Inard (2014) conducted a parametric study creating three samples varying the
518 thickness of the vegetation (Sample 1= 0.40 m; Sample 2 = 0.35 m; Sample 3 = 0.40 m) and
519 substrate (Sample 1= 0.07 m; Sample 2 = 0.15 m; Sample 3 = 0.09 m). LAD and *Sr* values were
520 kept constant (LAD = 2 m³ m⁻³; *Sr* = 4 m³ m⁻³). The study showed that more heat was

521 transmitted to the building wall in Sample 1 and decreased in Sample 2 due to greater thickness.
522 Finally, an increase in latent heat was obtained for Sample 3.

523 It is tough to compare the mere contribution of the substrate with other studies as substrates are
524 highly varied in type, dimensions, and thermophysical properties. However, thicker and more
525 continuous substrate layers are believed to lead to major temperature reductions as more
526 evaporation occurs. Besides, as more thermal mass is present, substrates could host larger plant
527 species which can subsequently contain greater *LAD* (Arengi, Perra and Caffi, 2021).

528 **6 Conclusions and future perspectives**

529 This work evaluated a hygrothermal model for predicting the temperature reduction at the
530 building wall's surface provided by green wall-test panels (GW-TPs). A 3D multiphysics model
531 coupling heat and mass transfer and heat sink terms to account for the evapotranspiration effect
532 was integrated into Comsol Multiphysics[®]. Such an approach offers unprecedented insights into
533 the complex fluid dynamics and heat transfer mechanisms that govern the behavior of green
534 walls and the application of more boundary conditions. The model was validated under steady-
535 state conditions against experiments performed in a climate chamber with sensors placed at
536 multiple positions across the entire wall composition, providing a much sounder base for
537 validation. In addition, natural convection occurring in the enclosed air cavity was modeled,
538 and its contribution was quantified. Finally, the masonry's surface temperature was paid special
539 attention to compare the temperature reduction in a bare wall vs. a green wall scenario.

540 Our model predicted that under evapotranspiration conditions, including the insulation and air
541 cavity effects, the four GW-TPs could decrease the masonry's surface temperature in the range
542 of 0.89 to 1.14 °C (0.97 ± 0.11 SD °C) and a mean temperature reduction at the substrate's rear
543 of 0.57 ± 0.15 SD °C. The temperature reduction attributed to the evapotranspiration effect
544 under this scenario was 0.31 ± 0.20 SD °C (mean for all GW=TPs), which represents, on

545 average, 30% of the total temperature reduction at the masonry's surface. The air cavity
546 accounted for 60.7 ± 0.09 % (on average) at the same location. Such values are lower than
547 previous hygrothermal models' predictions (> 10 °C), which can be explained as solar radiation
548 was not replicated in our experiments nor used as a boundary condition in the model. However,
549 we demonstrated that if considered, our model predicts a temperature reduction at the masonry's
550 surface of almost 10 °C compared to the bare wall scenario. These results align with values
551 reported in previous hygrothermal models.

552 The evaporation effect, which received special attention, was divided into the plant's
553 transpiration and the substrate's evaporation. The former was expressed regarding biological
554 parameters like leaf area density (LAD) and the latter in terms of the substrate's water content
555 (Sr). Heat sink terms were applied in the model to represent the evapotranspiration effect which
556 deducted heat from the global heat source.

557 Leaf area density (LAD) and the substrate's water saturation (Sr) directly impacted the ability
558 of the GW to reduce the building's surface temperature. Higher values in both parameters were
559 translated into higher temperature reduction values at the masonry's surface. Future work
560 should employ innovative techniques to determine the real values of LAD that can be further
561 used in the simulation. Hence, seeking their optimal combination to achieve maximum
562 temperature reduction in future work is crucial.

563 **7 Declaration of Competing Interest**

564 The authors have no declarations of interest.

565

566 **8 Acknowledgements**

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697 Appendix A

698 Green Wall-Test Panels Description

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700 Each GW-TP dimension was 0.52 m (height) x 0.61 m (width) with variable thickness according
701 to the substrate. The GW-TPs were mounted on the outer leaf of an insulated cavity wall (CW)
702 that serves as a control representing a building wall. The latter consisted of (from outer to inner
703 side) masonry (0.102 m), air gap (0.03 m), rockwool (insulation) (0.05 m), and Thermobrick
704 (0.09 m).

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706

707 *Figure A.1. Green wall-test panels (GW-TPs) assessed in the climate chamber. (a) PANEL A; (b) PANEL B; (c) PANEL C;*
708 *(d) PANEL D.*

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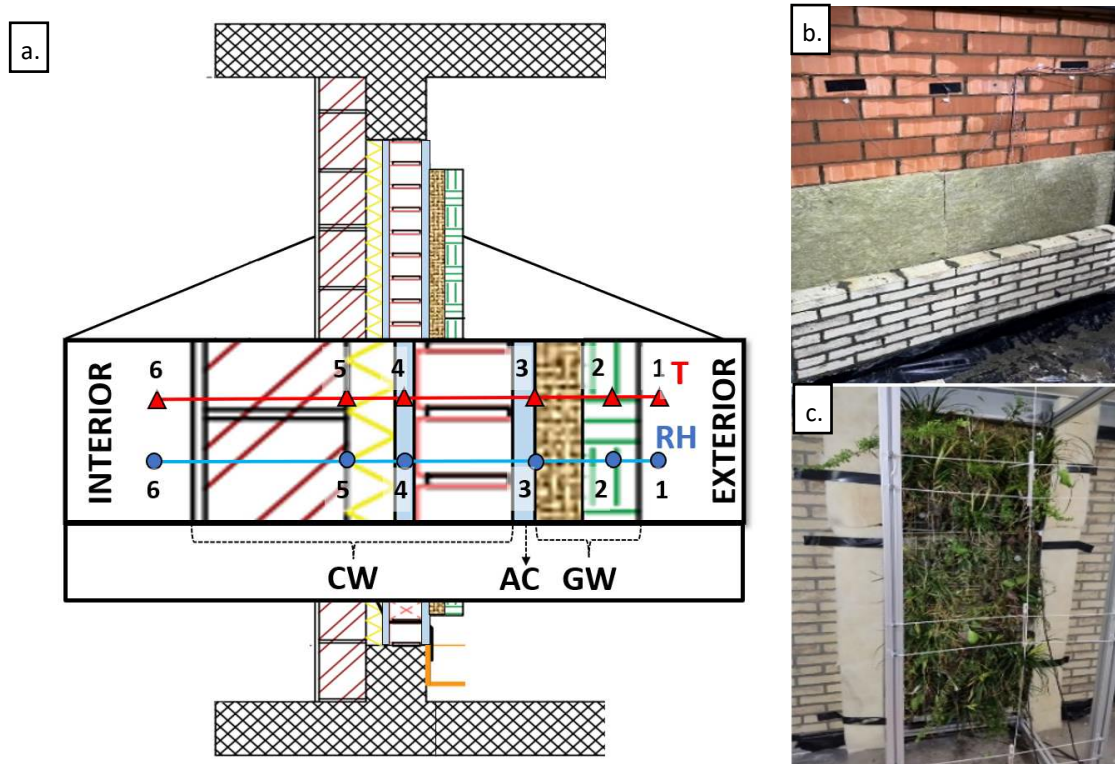
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713 Appendix B
 714 Climate Chamber

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 716 The chamber was operated by the Energy and Materials in Infrastructure and Buildings (EMIB)
 717 Research Group of the Department of Applied Engineering, University of Antwerp.



718
 719 *Figure B.1. (a) Side view of the climate chamber with the green wall (GW) compartment, which is separated from the*
 720 *insulated cavity wall (CW) by an air cavity (AC). The exterior of the CW is the masonry's surface, whereas its interior*
 721 *corresponds to the Thermobrick surface. Six sensors to measure temperature (red triangles), i.e., T1 – T6, and relative*
 722 *humidity (blue points) RH1 – RH6, are purposely located across the setup (from the exterior to the interior of the setup),*
 723 *each placed in the following order: (1) exterior room; (2) middle of the vegetation; (3) rear of the substrate; (4) middle of the*
 724 *inner air cavity between masonry and Rockwool; (5) backside of the Rockwool; (6) interior room. (b) Photograph of the*
 725 *different materials that constitute the insulated cavity wall (CW). (c) Photograph of the green wall test panels (GW-TP)*
 726 *evaluated in the climate chamber, mounted on the exterior or outer leaf of the insulated air cavity wall (CW).*

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735 Appendix C

736 Thermophysical properties.

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738 *Table C.1. Thermophysical properties of the layers constituting the setup evaluated in the present work.*

Section	Layer	Thermal conductivity	Specific heat capacity	Density	Vapor resistance factor	Reference
		(k)	(C_p)	(ρ)	(μ)	
		$W m^{-1} K^{-1}$	$J kg^{-1} K^{-1}$	$kg m^{-3}$	(-)	
Green wall-test panel (GW)	Vegetation	0.5	2252	656	1	(Jayalakshmy and Philip, 2010)
	Substrate	1.5	2085	1500	1.4	(Arengi, Perra and Caffi, 2021).
Air cavity	Air	0.025	1006	1.23	1	(Date, 2012)
Insulated cavity wall (CW)	Masonry	1.61	800	1800	13	(Wienerberger, 2021a)
	Air	0.025	1006	1.23	1	
	Rockwool	0.035	1030	35	1.4	(ROCKWOOL, 2022)
	Thermobrick	0.29	1000	100	13	(Wienerberger, 2021b)

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